

1 **ARRANGEMENT AND METHOD FOR MAINTAINING**
2 **A MINIMUM FLOW VELOCITY IN THE COOLANT RETURN OF A**
3 **MACHINE TOOL COOLANT FILTRATION SYSTEM**
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6 **Background of the Invention**

7 This invention concerns central filtration apparatus supplying filtered coolant to a
8 plurality of machine tools. Machine tools typically are supplied with a flow of machining
9 coolant which is directed through nozzles at the cutting tools and machined parts to lubricate and
10 cool the cutting tools as the machining process proceeds.

11 The filtration apparatus is provided to filter the returned coolant and supply an
12 adequate flow to all of the machine tools at a proper pressure.

13 The used coolant drains down into an individual sump located beneath each
14 machine tool, which is equipped with a float control and pump to pump the used coolant back
15 through a common return pipe to the filtration apparatus.

16 The coolant also washes away cutting chips and other solid material generated by
17 the machining process, such as grinding swarf. Coolant and entrained solids flowing through the
18 return pipe must be under sufficient pressure in order to insure that a sufficient minimum flow
19 velocity is maintained along the length of the return pipe such that the solids are carried along
20 with the coolant in the return pipe. If the flow velocity is too low, the entrained solids may settle
21 out and create blockages.

22 Different types of machining operations produce solids of different densities such
23 that different minimum velocities must be maintained according to the type of solids being
24 generated.

25 Piping of different sizes and configurations used for the coolant return also affect

1 the minimum flow velocities necessary. In large scale installations with large flow volumes and
2 large diameter pipes, the losses are low such that it is easier to maintain sufficient velocities due
3 to the mass of the flowing liquid. Sloping of the return piping may be adequate to maintain the
4 proper velocities.

5 However, in smaller installations, particularly with multiple turns and other
6 restrictions, maintaining sufficient flow velocities is more difficult due to the much higher losses.

7 Another complication, particularly in small installations, is the considerable
8 variations in system operating conditions which occur, such as during start up and the substantial
9 effects of having one or more machine tools not operating at any given time. There may occur
10 system conditions where one or more the machine tools are not pumping any dirty coolant into
11 the return so that sufficient coolant flow in the return is not present to maintain the required
12 minimum velocity of the return flow.

13 While clean coolant could simply be directed into the return piping via a bypass
14 connection to maintain the flow velocities above the minimum level required, this would
15 increase the required capacity of the central filtration apparatus as a significant portion of the
16 filtered coolant would not be available for use by the machine tools.

17 Also, if a large flow of clean coolant is diverted into the return piping, elevated
18 pressures therein requires increased sump pump output since dirty coolant must be introduced
19 into the return piping at an elevated pressure. Larger piping networks would also be necessitated,
20 further increasing the cost of the installation.

21 In order to deal with these variable conditions, the practice has in the past
22 typically been to collect the flow from all of the sumps in a large auxiliary tank, with a separate

1 additional pump used to pump coolant collected in the auxiliary tank back to the filtration
2 apparatus. This additional equipment is a significant part of the cost of the system, particularly
3 as a back up pump is often specified to be available if the first pump fails in order to reach a
4 higher operating reliability.

5 Accordingly, it is the object of the present invention to provide an arrangement for
6 insuring minimum flow velocities in the return piping for machine tool coolant filtering systems
7 where maintaining a minimum flow velocity is difficult, over a wide range of varying system
8 operating conditions.

9 It is another object of the invention to provide such an arrangement which does
10 not entail any increase in the capacity of the filtering apparatus or sump pump sizes and without
11 requiring any additional tanks or pumps.

12 13 Summary of the Invention

14 These and other objects of the present invention which will become apparent upon
15 a reading of the following specification and claims are achieved by providing a branch pipe from
16 the clean coolant supply to the return piping via a pressure control valving arrangement which
17 insures that a make up flow of clean coolant is introduced into the return piping just sufficient to
18 maintain the minimum flow velocity in the return piping.

19 The pressure control valving arrangement includes a pressure reducing valve
20 connected upstream of the return piping which is set to maintain a predetermined calculated or
21 empirically determined minimum pressure in the return line which is necessary to maintain
22 minimum flow velocities in the return piping of the particular system.

1 Thus, only the minimum diversion of clean coolant into the return line occurs to
2 automatically introduce only that clean coolant flow required to just make up any temporary
3 difference in return flow from the machine tools.

4 The pressure control valving arrangement further includes a pressure maintaining
5 valve upstream of the pressure reducing valve set to close when upstream pressure in the coolant
6 supply line drops to the minimum pressure required to properly supply coolant to the machine
7 tools.

8 An excessive diversion of clean coolant such as to create an insufficient pressure
9 of coolant to the machine tools is avoided by closing of the pressure maintaining valve to prevent
10 flow of clean coolant into the return line via the pressure reducing valve even if the return piping
11 is unpressurized at that time. This will not create a blockage problem since no dirty coolant is
12 being pumped into the return piping when low pressure conditions exist in the return piping, as
13 when the sumps are being filled at start up. Since no dirty coolant is being pumped into the
14 return piping at this time, no clog up can occur.

15 16 Description of the Drawings

17 Figure 1 is a diagrammatic representation of a machine tool filtration system and a
18 machine tool installation connected thereto.

19 Figure 2 is an enlarged diagrammatic representation of the system components
20 associated with each machine tool components.

21 22 Detailed Description

1 In the following detailed description, certain specific terminology will be
2 employed for the sake of clarity and a particular embodiment described in accordance with the
3 requirements of 35 USC 112, but it is to be understood that the same is not intended to be
4 limiting and should not be so construed inasmuch as the invention is capable of taking many
5 forms and variations within the scope of the appended claims.

6 Referring to the drawings, and particularly Figure 1, a centralized filtration
7 apparatus 10 produces filtered machine tool coolant which is supplied under pressure to a
8 machine tool installation 12 by a centrifugal pump 14 via supply piping 16. The filtration
9 apparatus 10 may be of a type in wide use for this application in which dirty coolant is introduced
10 into a tank 18 through a return pipe 20.

11 This contaminated coolant containing machining debris solids is drawn through a
12 belt media periodically indexed over a perforate plate covering a vacuum chamber (not shown).
13 Operation of the pump 14 connected to the vacuum chamber creates a vacuum causing the
14 coolant to be drawn through the media to thereby be cleaned, which then is recirculated to the
15 machine tools via piping 16.

16 The pump 14 and filter apparatus 10 are sized to provide a flow of clean coolant
17 to the machine tools 20A, B, C, D adequate for the particular machining processes, i.e., 400
18 gallons per minute at 55 psi for example. Each machine tool would typically require 100 gallons
19 per minute at 45 psi in this example.

20 This type of apparatus is well known in the industry and does not in itself
21 comprise the present invention such that further details are not here set out.

22 The machine tool installation 12 is comprised of several machine tools 22A, B, C,

1 D, each supplied with filtered coolant via individual branches included in the supply piping 16.

2 The coolant is used by being directed at the workpiece surface being machined with cutting tools
3 to facilitate the machining process.

4 The coolant flushes away the chips, and other solid debris such as grinding swarf,
5 etc., and the coolant and entrained solids drain down into an associated sump 24 over shed plates
6 25 where they are collected (Figure 2).

7 Each sump 24 has an associated sump pump 26 turned on and off by operation of
8 a level control device 28 maintaining a predetermined level of drained coolant in the sump 24.

9 Each sump pump 30 pumps the dirty coolant out of the sump 24 via an outlet pipe
10 32 to a return piping manifold 34 where the flow from each sump enters the return piping 20 to
11 be directed back to the filtration apparatus 10.

12 This arrangement is typical of centralized coolant filtration systems supplying a
13 plurality of machine tools.

14 According to the concept of the present invention, a make up flow of clean
15 coolant is caused to be controllably introduced into the return pipe 20 via a bypass line 36
16 connected to the main supply piping 16 and to the return piping manifold 34 via a pressure
17 control valving arrangement constituted by a pressure sustaining valve 38 and a pressure
18 reducing valve 40.

19 The pressure reducing valve 40 is of a commercially available type which
20 automatically maintains an adjustably preset downstream pressure regardless of variations in the
21 inlet pressure or flow rates. The pressure to be maintained is predetermined for any given
22 installation as by calculation and/or testing is that pressure which will produce the minimum flow

1 velocity of the coolant in the return piping necessary for that installation. That is, the pressure
2 required at the return header 34 will vary with the length and diameter of the return piping 20 due
3 to the resulting variations in the pressure losses due to friction, turns and other restrictions, to
4 maintain the minimum velocity in the return piping 20. Whatever that pressure is determined to
5 be, the valve 40 is preset to maintain that pressure level. This would typically be at a pressure of
6 4 or 5 psi in a typical system.

7 Thus, if the combined flow from all of the sump pumps 26 is not sufficient to
8 create the necessary pressure at the header 34, a make up flow of clean coolant is automatically
9 introduced at the header 34 of a volume just sufficient to increase the pressure level to that
10 predetermined to be required.

11 The pressure reducing valve 40 is used to precisely control the make up flow to
12 avoid excessive diversion of clean coolant into the return piping 20 which might otherwise occur.

13 The pressure maintaining valve 38 prevents excessive diversion of coolant during
14 high flow demand with all of the machines running, when low pressure conditions in the return
15 piping 16 prevails.

16 For example, at start up, maximum flow to the machine tools 22 may occur if all
17 of the tools A, B, C, D are turned on. At the same time, each sump 24 needs to be filled before
18 dirty coolant flow begins to enter the return piping 20 so that very low or zero pressure exists in
19 the return piping. Thus, pressure reducing valve 40 will open to a maximum opening to attempt
20 to pressurize the return piping 20. Thus, a large flow of clean coolant in the return piping 20
21 would occur at initial start up or possibly at other times, and pressure to the machine tools A, B,
22 C, D might fall below the level necessary to provide adequate flow thereto, i.e., 45 psi at each

1 machine tool 22.

2 A make up flow in the return piping 20 is not necessary at that time since no
3 entrained solids are yet being introduced therein so that clogging cannot occur.

4 The pressure maintaining valve 38 is therefore set to close at the minimum
5 pressure required in the supply piping 16 to insure proper flow to all of the machine tools, i.e., 45
6 psi. Thus, coolant flow is prevented from reaching the pressure reducing valve 40, and thus no
7 make up flow of clean coolant is diverted to the return piping 20.

8 After normal conditions are reached and pressure in the supply piping 16 again
9 exceeds 45 psi, pressure maintaining valve 38 will again open to allow the volume make up flow
10 as necessary to compensate for fluctuations in the total flow of coolant in the return piping 20.

11 The cost of the two valves 38, 40 (which are pressure operated and do not require
12 wiring or controls), is much lower than increasing sump pump sizes or than adding an additional
13 pump and the auxiliary tank, and are quite reliable over an extended service life.

14 The arrangement can be tailored to a wide variety of installation configurations,
15 and does not utilize a significant fraction of the capacity of the filter apparatus 10 so that the
16 filter apparatus need be sized to provide only that capacity necessary to serve the machine tools.